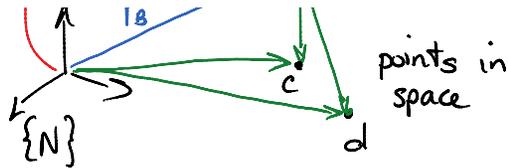


$$T_B^N = \begin{bmatrix} R_B^N & p_B^N \\ 000 & 1 \end{bmatrix} \in SE(3)$$

where $R_B^N \in SO(3)$



$$\therefore (R_N^B)^{-1} = (R_N^B)^T = R_B^N = \begin{bmatrix} \hat{n}_B^N & \hat{t}_B^N & \hat{o}_B^N \end{bmatrix}$$

$\hat{n}, \hat{t}, \hat{o}$ are orthonormal and

right-handed, i.e. $\text{Det}(R) = +1$

R is parameterized with three or more parameters, often a unit quaternion $[e_0 e_1 e_2 e_3]$, where

$$e_0^2 + e_1^2 + e_2^2 + e_3^2 = 1$$

$$(T_B^N)^{-1} = \begin{bmatrix} (R_B^N)^T & -(R_B^N)^T p_B^N \\ 000 & 1 \end{bmatrix} = T_N^B$$

Transforming a point c from frame $\{B\}$ to frame $\{N\}$

$$T_B^N c^B = c^N, \text{ where } c^B \text{ \& } c^N \text{ are in homogenous form, i.e. } [c_x \ c_y \ c_z \ 1]^T$$

Transforming a direction from frame $\{B\}$ to frame $\{N\}$

unit vectors & vector differences represent directions.

$$T_B^N d^B - {}_B^N T c^B = T_B^N (d^B - c^B) = T_B^N \begin{bmatrix} d_x - c_x \\ d_y - c_y \\ d_z - c_z \\ 1 - 1 \end{bmatrix}$$

$$\begin{bmatrix} R_B^N & p_B^N \\ \vdots & \vdots \end{bmatrix} \begin{bmatrix} \Delta x \\ \Delta y \\ \Delta z \end{bmatrix}^B = \begin{bmatrix} R_B^N \Delta^B \\ \text{---} \end{bmatrix}$$

$$\begin{bmatrix} R_B & P_B \\ 0 & 1 \end{bmatrix} \begin{bmatrix} \Delta y \\ \Delta z \\ 0 \end{bmatrix} = \begin{bmatrix} R_B^N \Delta^B \\ 0 \end{bmatrix}$$

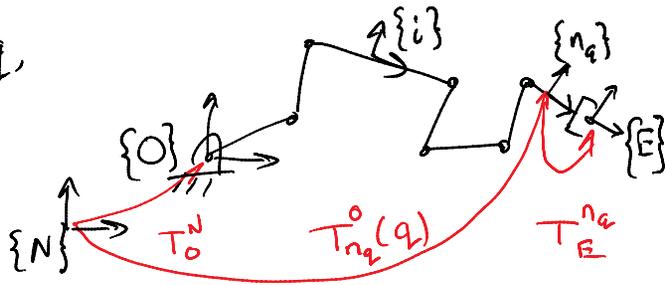
So, transforming directions is simply vector rotation.

If you simply want to determine the components of a vel, ang. vel, force, or moment w.r.t. another frame, use R .

$$R_B^N [v^B \parallel \omega^B \parallel f^B \parallel \tau^B] = [v^N \parallel \omega^N \parallel f^N \parallel \tau^N]$$

Forward Position Kinematic Map

Given joint angles, q ,
determine T_E^0



$$T_0^N T_1^0(q_1) \dots T_{n_q}^{n_{q-1}}(q_{n_q}) T_E^{n_q} = T_E^N(q)$$

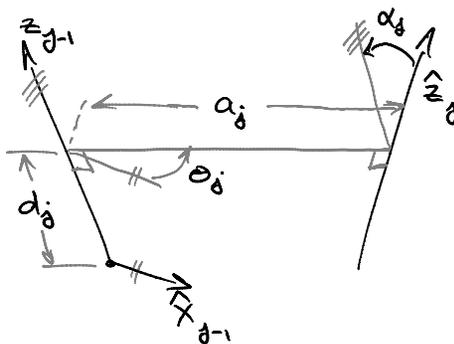
Methods:

Denavit-Hartenburg (std or mod)

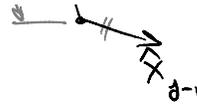
Product of Exponentials

Standard D-H parameters

$$T_i^{i-1} = \begin{bmatrix} c_{\theta} & -s_{\theta} c_{\alpha} & s_{\theta} s_{\alpha} & a_{\theta} c_{\theta} \\ s_{\theta} & c_{\theta} c_{\alpha} & -c_{\theta} s_{\alpha} & a_{\theta} s_{\theta} \end{bmatrix}$$



$$T_{j-1}^j = \begin{bmatrix} c_{\theta_j} & -s_{\theta_j} c_{d_j} & s_{\theta_j} s_{d_j} & a_j c_{\theta_j} \\ s_{\theta_j} & c_{\theta_j} c_{d_j} & -c_{\theta_j} s_{d_j} & a_j s_{\theta_j} \\ 0 & s_{d_j} & c_{d_j} & d_j \\ 0 & 0 & 0 & 1 \end{bmatrix}$$



Inverse Position Kinematic Map

Given $T_{E, \text{target}}^N$, determine q

Methods:

Closed form: expand and solve

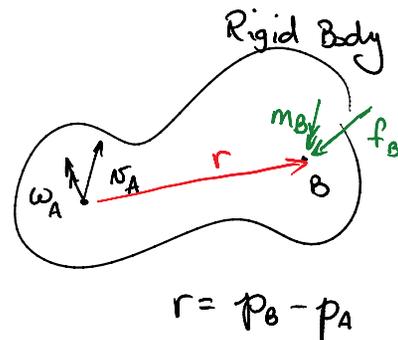
$$\underbrace{T_1^0(q_1) \dots T_{n_2}^{n_2-1}(q_{n_2})}_{\text{multi-linear functions of } \sin(q_i) \text{ \& } \cos(q_i)} = \underbrace{T_N^0 T_{E, \text{target}}^N T_{n_2}^E}_{\text{given fixed values}}$$

Iterative:

Solve simultaneous nonlinear equations.

Transforming body wrenches & twists

Let $\nu = \begin{bmatrix} \omega \\ v \end{bmatrix}_{6 \times 1}$ be the twist of a point on a rigid body.



If ν_A is known, what is ν_B ?

$$\omega_B = \omega_A + \omega_A \times r = \omega_A - r \times \omega_A$$

$$\omega_B = \omega_A$$

$$\begin{bmatrix} 0 & -r_2 & r_1 \end{bmatrix}$$

$$\omega_B = \omega_A$$

$$\text{Let } P(r) = \begin{bmatrix} 0 & -r_z & r_y \\ r_z & 0 & -r_x \\ -r_y & r_x & 0 \end{bmatrix}$$

$$v_B = \begin{bmatrix} N_B \\ \omega_B \end{bmatrix} = \begin{bmatrix} I_{3 \times 3} & P^T(r) \\ 0_{3 \times 3} & I_{3 \times 3} \end{bmatrix} \begin{bmatrix} N_A \\ \omega_A \end{bmatrix}$$

Let $g = \begin{bmatrix} f \\ m \end{bmatrix}$ be the wrench of a force f with a given line of action, l .

$$g = \begin{bmatrix} f_B \\ m_B \end{bmatrix} \text{ where } l \text{ contains } B.$$

We want to determine g_A such that it has the same effect on the rigid body as g_B

$$\left. \begin{array}{l} f_A = f_B \\ r_A = m_B + r \times f_B \end{array} \right\} \Rightarrow g_A = \begin{bmatrix} I_{3 \times 3} & 0_{3 \times 3} \\ P(r) & I_{3 \times 3} \end{bmatrix} g_B$$

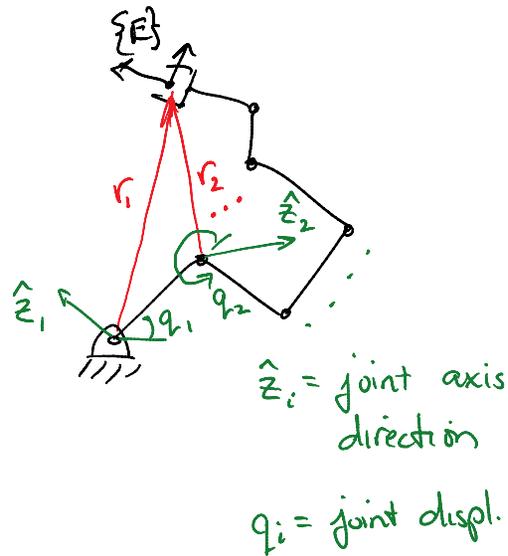
If there are coordinate frames $\{A\}$ and $\{B\}$ on the rigid body and we want $g_A \neq v_A$ expressed in $\{A\}$ and $g_B \neq v_B$ expressed in $\{B\}$, then we have:

$$\left. \begin{array}{l} g_A = \begin{bmatrix} R_B^A & 0_{3 \times 3} \\ 0_{3 \times 3} & R_B^A \end{bmatrix} \begin{bmatrix} I & 0 \\ P(r) & I \end{bmatrix} g_B \\ v_B = \begin{bmatrix} R_A^B & 0 \\ 0 & R_A^B \end{bmatrix} \begin{bmatrix} I & P^T(r) \\ 0 & I \end{bmatrix} v_A^A \end{array} \right\} \text{ where } R_B^A \text{ is } \begin{bmatrix} \hat{n}_B^A & \hat{t}_B^A & \hat{o}_B^A \end{bmatrix}$$

Manipulator Jacobians

$$J\dot{q} = v_E \quad \& \quad J^T \tau = g_E$$

where E denotes the end effector frame.



$$J_E(q) = \begin{bmatrix} J_{P_E} \\ J_{\phi_E} \end{bmatrix}$$

$$J_{P_E} = \begin{bmatrix} P(r_i) \hat{z}_1 & \hat{z}_2 & \dots \end{bmatrix}$$

revolute
prism.

$$J_{\phi_E} = \begin{bmatrix} \hat{z}_1 & 0 & \dots \end{bmatrix}$$

rev.
prism.

Singular Configs

It is desirable for control, that $J_E(q)\dot{q} = v_E$ has a solution for every $v_E \in \mathbb{R}^6$.

If $\exists v_E \in \mathbb{R}^6 \ni \dot{q} \nexists$ satisfying $J_E(q)\dot{q} = v_E$, the q is a singular configuration.

If for some q , \exists a set of q satisfying $J_E(q)\dot{q} = v_E$, then the manipulator is redundant.

Manipulator dynamics

$$\tau = M(q)\ddot{q} + V(q, \dot{q}) + \underbrace{G(q)}_{\text{gravity}} + \underbrace{F(q, \dot{q})}_{\text{end effector wrench}} + J_E^T(q) g_E$$

$$\tau = \underbrace{M(q)}_{\text{P.D. mass matrix}} \ddot{q} + \underbrace{V(q, \dot{q})}_{\text{Coriolis centripetal}} + \underbrace{G(q)}_{\text{friction}} + F(q, \dot{q}) + J_E^T(q) g_E$$

Linear equations $Ax = y$ $A \in \mathbb{R}^{m \times n}$ $x \in \mathbb{R}^n$ $y \in \mathbb{R}^m$

Rank(A) = # of l.i. rows \leftarrow always the same.
 = # of l.i. columns \leftarrow

Null Space of A = $\mathcal{N}(A) = \{x \mid Ax = 0, x \in \mathbb{R}^n\}$

Left Null Space of A = Null Space of A^T

Column space of A = $\mathcal{R}(A) = \{y \mid Ax = y, x \in \mathbb{R}^n\}$

Row space of A = $\{x \mid A^T y = x, y \in \mathbb{R}^m\}$

Row space of A = $\mathcal{R}(A^T)$

Dimension($\mathcal{R}(A^T)$) = Dimension($\mathcal{R}(A)$) ALWAYS

Solution Existence

$Ax = y$ has a solution if $y \in \mathcal{R}(A)$

$x = \underbrace{A^+}_{\text{pseudo-inverse}} y + \underbrace{N(A)}_{\text{basis for } \mathcal{N}(A)} \alpha$
 α arbitrary vector of length = $\text{Dim}(\mathcal{N}(A))$

IF $\mathcal{N}(A) = 0$, then solution is unique.

If A is square & full rank, then $A^+ = A^{-1}$
and $\mathcal{N}(A) = \mathbf{0}$.

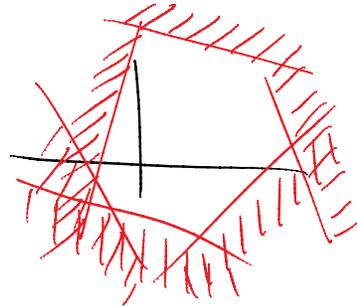
Linear Inequalities

$Ax \geq y \Rightarrow$ a polytope

polytope may be:

empty

nonempty \rightarrow compact convex set
 \rightarrow unbounded convex set



A necessary condition for the polytope to be empty is $m > n$.

Phase 1 of the Simplex method can determine if the polytope is empty, and if not, it will find a point in the polytope.